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Fatigue life modelling of railway prestressed concrete sleepers

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12 **Abstract**

13 The railway sleepers, which transfer wheel loads to the formation, are an important component of
14 railway track systems. Prestressed concrete is the most commonly used type of railway sleeper around
15 world. Crushing is a common problem on concrete sleepers, which include excessive flexural, shear,
16 and bond stresses. The most causes of crushing in prestressed concrete sleepers are dynamic loads.
17 However, accumulated damage due to cyclic loads can also cause crushing. Much previous research
18 has investigated the impact load characteristics and the ultimate load capacity of prestressed concrete
19 railway sleepers. There is a gap in the knowledge in fatigue failure for prestressed concrete sleepers.
20 This study presents new results of extensive numerical and analytical investigations aimed at predicting
21 fatigue lives under cyclic loads. A numerical study validated by 30 full-scale experimental tests is
22 executed to assess fatigue performance, while theoretical fatigue analysis methods based on S-N curve
23 and Miner linear cumulative damage are introduced for benchmarking. This paper presents a remaining
24 fatigue life assessment for prestressed concrete sleepers and contrasts with the theoretical results.
25 Parametric studies discuss the effect of support conditions, dynamic load distribution, and track
26 stiffnesses on prestressed concrete sleepers. This paper highlights the rationales associated with the
27 development of fatigue limit state. The outcome of this paper will provide design flexibility and
28 improve railway sleeper maintenance and inspection criteria.

29 **Keywords:** prestressed concrete sleeper, fatigue, cyclic load, finite element method (FEM), numerical
30 analysis

31 **1 Introduction**

32 Railway transportation provides safe, economical, and comfortable transport for either passengers or
33 goods [1, 2]. An important component of railway systems are railway sleepers. The main functions of
34 railway sleepers are distributing axle loads from rails to the substructures of railway systems and
35 holding the rails at the proper gauge [1, 3]. Prestressed concrete railway sleeper is the most commonly
36 used type of sleeper around world with its economic cost, long service life, and sound structural
37 performance. However, prestressed concrete sleepers can still be damaged due to flexural, shear, or

38 bond stresses. In long-term service, fatigue load heavily influences progressive cracking. Fatigue
39 failure can be defined as structural failure below the stress limit under cyclic loading [4]. A worldwide
40 survey of most critical concrete sleeper failures was conducted by W. Ferdous, A. Manalo (2014), and
41 the results indicated fatigue is one of the top 5 most critical causes of failure for concrete sleepers [5].
42 Fatigue failure can be defined as a failure happens below the stress limit of a material when it has been
43 applied to repeated loads. In concrete sleepers, crushing or cracking under compressive stress can be
44 considered ordinarily as the failure criterion to define fatigue state in design process. A global trend
45 that the increases in traffic flow and train tonnage cause accumulated damage due to cyclic loads,
46 which might cause railway sleeper failure in long-term service life.

47 The permissible stress or allowable stress design concept for prestressed concrete sleepers is used in
48 many countries [6]. This method is based on the permissible stress of materials and a load factor is
49 used to increase the static axle load to incorporate dynamic effects [7]. However, the permissible stress
50 method may underestimate the material strength, while dynamic loads are not considered thoroughly.
51 A concept based on a probabilistic model of the load resistance, limit state method, is developed for
52 railway sleepers. It applies the magnitude of factors may be varied so that they may be used either with
53 the plastic conditions in the ultimate state or with the more elastic stress range at working loads [8].
54 Ultimate limit state is the railway sleepers must be able to withstand the loads against failure. Fatigue
55 limit state is a concrete sleeper services under cyclic loading for years and the accumulated damage
56 progressively reaches failure [9].

57 Previous research by Wakui, H and Okuda, H (1997) found cracking of concrete sleepers developed
58 rapidly due to repeated impact loads, and it was suggested a fatigue limit state should be included in
59 concrete sleeper design [10]. Many experimental and analytical investigations regarding dynamic
60 behaviour of prestressed concrete railway sleepers have been conducted by Kaewunruen, S and
61 Remennikov, A.M, (2007, 2014, 2013) and their research provides amount of data for analysis of cyclic
62 loads and fatigue damage [11-13]. The fatigue loading tests were carried out by Rantala et al (2018) to
63 analyse the fatigue properties of the sleepers and the effect of fatigue on their stiffness [14]. Šimonová
64 et al presented a study on the influence of the age and level of concrete fatigue on prestressed railway
65 sleeper [15]. Maekawa, K. et al (2006) conducted the 3D fatigue analysis which predicts the fatigue
66 life of reinforced concrete slabs [16]. Zhang, C. et al (2019) developed 3D concrete T-shape girder in
67 order to investigate the fatigue performance of the carriageway plates of bridge [17]. These studies
68 provide a fundamental understanding of fatigue behaviour in concrete structures.

69 In this research, it is possible to better understand the fatigue life of the railway sleepers associated
70 with various service conditions. On this ground, there is clearly a need to carry out an investigation of
71 the fatigue life simulation. This article presents a numerical simulation of the fatigue life of prestressed
72 concrete sleepers. Finite element analysis was conducted to investigate the life cycle of prestressed
73 concrete sleepers. Chinese Type III prestressed concrete sleepers were used to study fatigue life. A
74 physical model of the sleeper is established and validated. The results show the effect of typical support
75 conditions, various impact loads, and different track stiffnesses on service life of the prestressed
76 concrete sleeper. A theoretical fatigue life assessment method is also introduced in order to validate
77 the numerical model. The findings presented in this study aim to develop a fatigue life assessment
78 model, which will eventually help railway engineers to better estimate the service life of railway
79 sleepers under various conditions.

80 **2 Fatigue life assessment method**

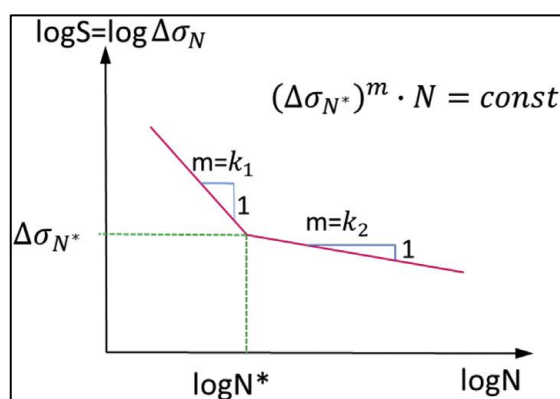
81 *2.1 Damage accumulation method*

82 Previous research has presented the fatigue life assessment method of concrete sleepers [9, 18]. The
 83 theoretical assessment method is based on the damage accumulation method and extending Miner's
 84 rule, and can be used to evaluate the fatigue life under constant amplitude cyclic loads. For each cyclic
 85 load, the critical tensile stress of the prestressing tendons of the prestressed concrete sleeper can be
 86 calculated and the fatigue life for the cyclic load can be determined. The cumulative damage index is
 87 given by:

$$88 \quad \sum D_i = \sum_i \frac{n(\Delta\sigma_i)}{N(\Delta\sigma_i)} \quad (1)$$

89 where $n(\Delta\sigma_i)$ is the applied number of cycles at a stress range $\Delta\sigma_i$; $N(\Delta\sigma_i)$ is the resisting number
 90 of cycles at a stress range $\Delta\sigma_i$.

91 The S-N curve is the relationship between the magnitude of an alternating stress and the number of
 92 cycles until there is failure for a selected material. The maximum applied number of cycles at an
 93 alternating stress can be determined and it is expressed as failure cycles (fatigue life) [19, 20]. The S-
 94 N curve for prestressing steel is shown in **Figure 1** and the failure cycles of prestressing steel under
 95 cyclic loads can be estimated by:



96
 97 **Figure 1.** S-N curve for prestressing steel [19]

98
 99 If $(\Delta\sigma > \Delta\sigma_{N^*})$

$$100 \quad \log N_f = \log N^* - k_1 [\log(\Delta\sigma) - \log(\Delta\sigma_{N^*})] \quad (2)$$

101 If $(\Delta\sigma < \Delta\sigma_{N^*})$

$$102 \quad \log N_f = \log N^* + k_2 [\log(\Delta\sigma_{N^*}) - \log(\Delta\sigma)] \quad (3)$$

103 where $\Delta\sigma$ is the stress range of the prestressing tendons; $\Delta\sigma_{N^*}$ is the stress range at N^* cycles; k_1 ,
 104 k_2 are the stress exponents. **Table 1** illustrates parameters of prestressing steel S-N curve. (Note: this
 105 S-N curve is able to define the type of prestressing tendons used in this research)

S-N curve of prestressing steel used for	Stress exponent		$\Delta\sigma_{N^*}$ at N^* cycles (Mpa)
	N^*	k_1	

Pre-tensioning

10⁶

5

9

185

106 **Table 1.** Parameters of S-N curve for prestressing steel [19]

107 2.2 *Fatigue life assessment*

108 To assess fatigue life, the cracking progression of the railway sleeper needs to be analysed. In the initial
109 stage, no cracking appears and the concrete stress at the bottom fibre can be calculated by:

$$110 \quad \sigma_{cF}^b = \frac{nA_{ps}\sigma_{se}}{A_t} + \frac{nA_{ps}\sigma_{se}e}{I_t}y_t \quad (4)$$

111 where σ_{se} is the effective stress per prestressing tendon; A_{ps} is the cross-section area of a tendon; e
112 is the eccentricity; n is the number of tendons; A_t is the transformed area of the sleeper; I_t is the
113 inertia moment of transformed section before cracking; y_t is the distance of the centroidal axis of
114 transformed area from soffit.

115 The cracking moment is calculated by:

$$116 \quad M_{cr} = I_t \frac{\sigma_{cF}^b + f_{cf}}{y_t} \quad (5)$$

117 where f_{cf} is the tensile strength of the sleeper.

118 As the cracking progresses, the neutral axis of the sleeper cross-section will change. Therefore, the
119 distance y_{CG} from the centre gravity of the effective transformed area to the top of the compressed
120 area needs to be determined by:

$$121 \quad y_{CG} = \text{root} \left[[S_{pcII} - n_e A'_{p3}(h - y_{cg} - d_3) - n_e A'_{p2}(h - y_{cg} - d_2) - n_e A'_{p1}(h - y_{cg} - d_1)], y_{cg} \right] \quad (6)$$

122 where the S_{pcII} is the first moment at the bottom fibre after cracking; A'_{pi} is the total area of the
123 prestressing tendons at layer i ; d_i is the distance from the prestressed tendons at layer i to the bottom
124 of tension area; n_e is the modular ratio.

125 The effective transformed section can be estimated using the transformed area of sleeper cross-section
126 A_{tII} :

$$127 \quad A_{tII} = A_{cII} + n_e A_p \quad (7)$$

128 where A_{cII} is the effective concrete area of sleeper cross-section.

129 The moment of inertia of the cracking section is given by:

$$130 \quad I_{cr} = I_{ccr} + n_e A'_{p3}(h - y_{cg} - d_3)^2 + n_e A'_{p2}(h - y_{cg} - d_2)^2 + n_e A'_{p1}(h - y_{cg} - d_1)^2 \quad (8)$$

131 where I_{ccr} is the inertia moment of effective centroid. (Note: the selected prestressed concrete sleeper
132 has three layers of prestressing tendons, the equation (6)-(8) can be also applied for other cross-section
133 of prestressed concrete sleepers.)

134 The effective inertia moment in the lifetime is calculated by:

$$I_{ef} = I_{cr} + (I_t - I_{cr}) \left(\frac{M_{cr}}{M_{max}} \right)^3 \quad (9)$$

136 where I_t is the inertia moment of transformed section before cracking; M_{cr} is the cracking moment;
 137 M_{max} is the maximum bending moment at the section under cyclic loads.

$$\Delta\sigma_{pt1} = n_e \frac{M_{max} - M_{min}}{I_{ef}} (h - y_{cg} - d_1) \quad (10)$$

139 where M_{min} is the minimum bending moment at the section under cyclic loads.

140 Using the output value of $\Delta\sigma_{pt1}$, the failure cycles of the prestressing tendons under constant cyclic
 141 loading can be estimated by Equation (2) or (3).

142 **3 Fatigue life simulation method**

143 *3.1 Fatigue analysis type in finite element method*

144 In this study, ANSYS Workbench has been used to investigate the fatigue life of prestressed concrete
 145 sleepers. The aim of fatigue analysis in this simulation is to ascertain capability of a material to survive
 146 from the cyclic loads during service life. In general, fatigue analysis can be categorised into Strain Life,
 147 Stress Life, and Fracture Mechanics. The Strain Life method focuses on crack initiation where the
 148 strain can be directly measured. This method deals with relatively low cycles (less than 10^5 cycles).
 149 Fracture Mechanics is used to determine crack growth. The time from crack initiation to the crack
 150 growing to a critical size can be calculated using this method. Sometimes, Fracture Mechanics (crack
 151 growth) is used with the Strain Life method (crack initiation) to determine total fatigue life.

152 In this research, the Stress Life method is adopted. This method is typically characterised by an
 153 empirical S-N curve as part of material definition of the sleeper model and modified by a variety of
 154 factors dealing with relatively high cycles (more than 10^5 cycles). It should be noted that fatigue life
 155 results using the stress life method show the available life for the given fatigue analysis without
 156 considering crack propagation or fracture. Therefore, the output results represent the cycles until the
 157 sleeper will fail due to fatigue [21, 22].

158 *3.2 Cyclic loading type*

159 The Constant amplitude, Proportional loading is applied in this research shown in **Figure 2**. Loading
 160 is of constant amplitude because only one set of FE stress results along with a loading ratio is required
 161 to calculate the alternating and mean values. Loading is proportional since only one set of FE results
 162 are needed (principal stress axes do not change over time). Common types of constant amplitude
 163 loading are fully reversed (apply a load, then apply an equal and opposite load; a load ratio of -1) and
 164 zero-based (apply a load then remove it; a load ratio of 0). Since loading is proportional, looking at a
 165 single set of FE results can identify critical fatigue locations. Likewise, since there are only two
 166 loadings, no cycle counting or cumulative damage calculations need to be done [21, 22].

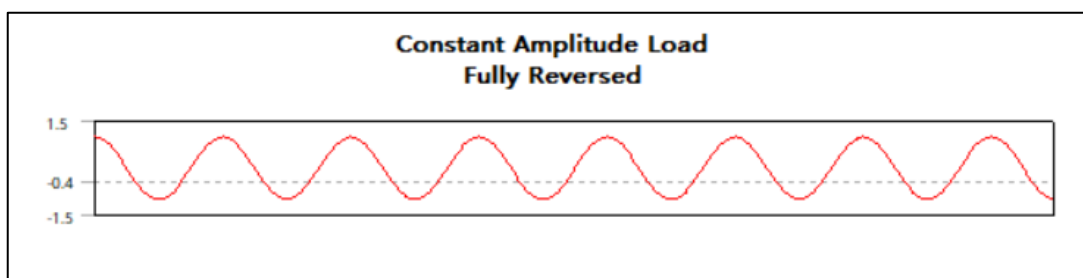


Figure 2. Constant amplitude load

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169 3.3 Equivalent alternating stress

170 In analysis of fatigue using the Stress Life method, the S-N curve is required to relate the fatigue to the
 171 stress state. Therefore, the equivalent alternating stress is used to query the S-N curve to determine the
 172 fatigue life. The equivalent alternating stress is the stress used to query the fatigue S-N curve after
 173 accounting for fatigue loading type, mean stress effects, multiaxial effects, and any other factors in
 174 fatigue analysis. The equivalent alternating stress can be determined as following steps:

- 175 1. Calculate the alternating mean stress tensor.
- 176 2. Collapse alternating and mean stress from tensor to scalar using selected stress component.
- 177 3. Calculate the Equivalent Alternating Stress using the desired empirical stress theory, as specified
 178 by the Mean Stress Theory property of the Fatigue Tool object.

179 Several empirical options can be selected including Gerber, Goodman, and Soderberg theories as
 180 shown below. According to these theories, the value reported as the equivalent alternating stress is used
 181 to query the fatigue life from the S-N curve. The equivalent alternating stress is the last calculated
 182 quantity before determining the fatigue life [21, 22].

183 *Gerber Equation:*

$$184 \frac{\sigma_{\text{Alternating}}}{S_{\text{Endurance Limit}}} + \left(\frac{\sigma_{\text{Mean}}}{S_{\text{Ultimate Strength}}} \right)^2 = 1 \quad (11)$$

185 *Goodman Equation:*

$$186 \frac{\sigma_{\text{Alternating}}}{S_{\text{Endurance Limit}}} + \frac{\sigma_{\text{Mean}}}{S_{\text{Ultimate Strength}}} = 1 \quad (12)$$

187 *Soderberg Equation:*

$$188 \frac{\sigma_{\text{Alternating}}}{S_{\text{Endurance Limit}}} + \frac{\sigma_{\text{Mean}}}{S_{\text{Yield Strength}}} = 1 \quad (13)$$

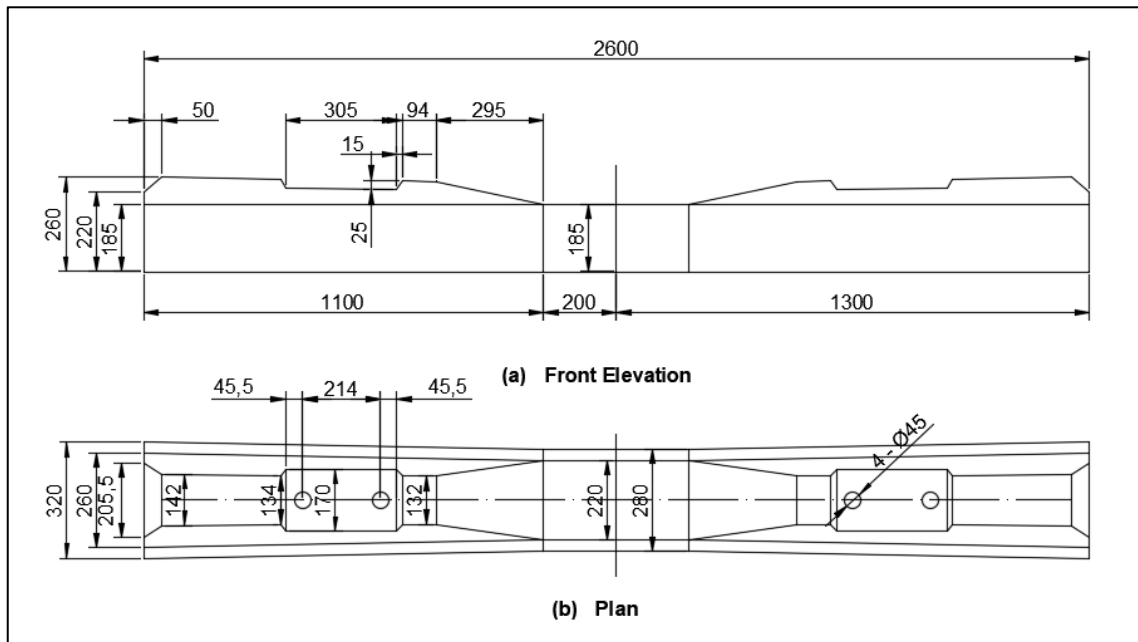
189 3.4 Fatigue life results

190 The simulation results of fatigue life are represented the number of cycles until the railway sleeper fails
 191 due to fatigue. In a Stress Life analysis with constant amplitude, if the equivalent alternating stress is
 192 lower than the lowest alternating stress defined in the S-N curve, the life at that point will be used [21,
 193 22].

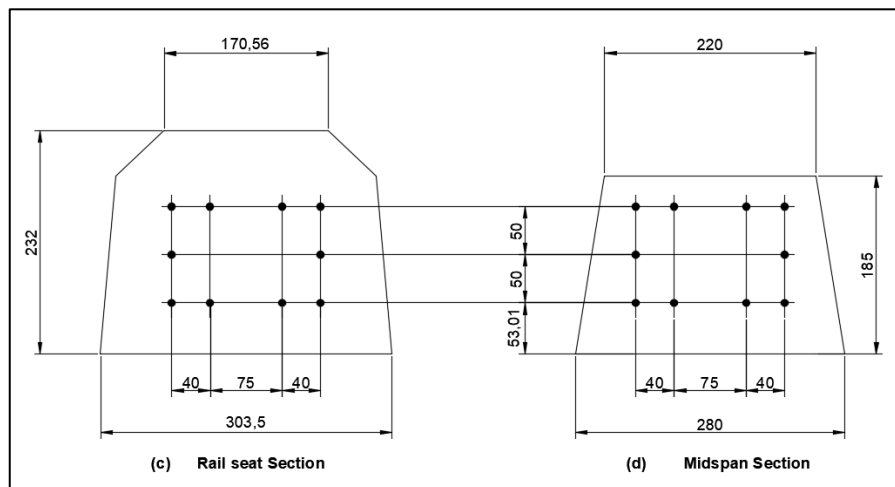
194 **4 Prestressed concrete sleeper details**

195 Chinese Type III prestressed concrete sleeper is used to study fatigue life. Material properties and
 196 section details are indicated in **Figure 3** and **Table 2** with the following parameters:

- 197 (1) Track gauge: 1435mm
 198 (2) Concrete strength: C60
 199 (3) Prestressing tendons: 10No. of 7mm dia. (tendon area: 384mm²)
 200 (4) Prestressing force: 420kN (pre-tensioning)



201



202

203 **Figure 3.** Chinese Type III prestressed concrete sleeper geometric details

Material properties	Basic variables	Value
Concrete	Mean compressive strength	65MPa
	Modulus of elasticity	33GPa
Prestressed wire	Yield strength	1570MPa

Modulus of elasticity

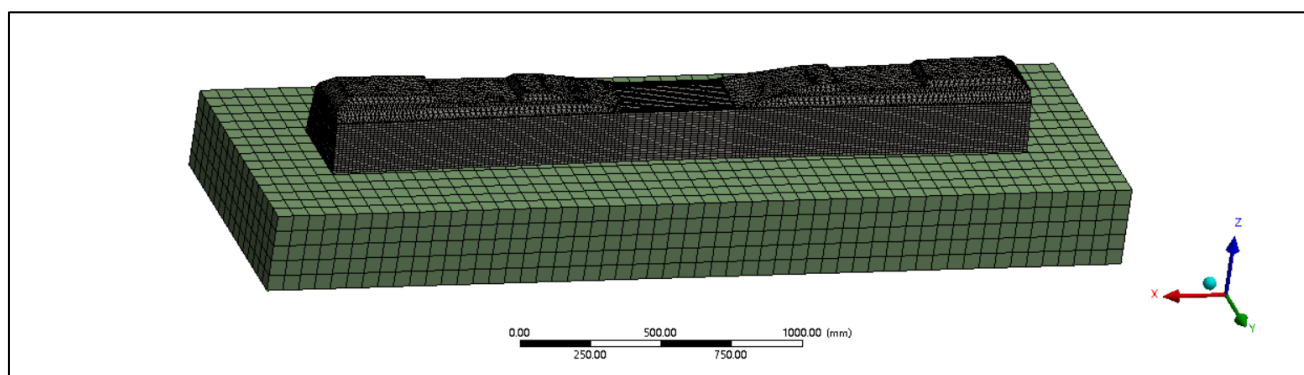
200GPa

204

Table 2. Material properties**5 Finite element model of the prestressed concrete sleeper**

206 The finite element model was developed to study fatigue life. The sleeper model used in this study is
 207 made up of 3D solid elements shown in **Figure 4**. The finite element model is composed of railway
 208 sleeper, prestressed tendons, and ballast. The concrete of sleeper is modelled as solid elements with
 209 most of these elements being 10-node tetrahedron elements, while the prestressed tendons are modelled
 210 as beam elements. The ballast is also modelled as solid elements. The material constitutive model of
 211 the FE sleeper model follows the experimental data presented in Section 5.1. It should be noted that
 212 the S-N curve used for prestressing tendons of the sleeper model is assumed in terms of **Table 1**.

213 The engineering properties used in the FE modelling are illustrated in **Table 3**. These properties were
 214 selected because they were identical to a particular type of concrete sleeper manufactured in China. In
 215 the model, concrete, prestressing tendons, and ballast (block support) are considered to be well adhere.
 216 The No-Separation contact type is used to simulate the constraints between concrete and prestressing
 217 tendons, which the bond slip and bursting are not considered. The contact type between the sleeper and
 218 ballast is Rough, which there is no sliding allowed. The bottom interface of tensionless ballast is set as
 219 fixed support and edges of ballast layer are set as free. The remaining boundary conditions at both rail
 220 seats have been set as hinges where the longitudinal and lateral displacements are restrained. In the
 221 simulation of prestressing force transfer, the Thermal Condition is used to define the temperature in
 222 the tendons, which the thermal load can be regarded as the prestressing force. After the prestressing
 223 force transferred, the cyclic load can be applied to the FE sleeper model for analysing fatigue life of
 224 the prestressed concrete sleeper. It should be noted that the size of ballast model is
 225 3200mm×900mm×300mm, which the selection follows the sleeper density with 1308 sleepers per km.
 226 The thickness of ballast layer is usually 250mm to 300mm, thus the 300mm thickness of ballast is
 227 accepted.



228

Figure 4. Finite element model of the Chinese Type III prestressed concrete sleeper

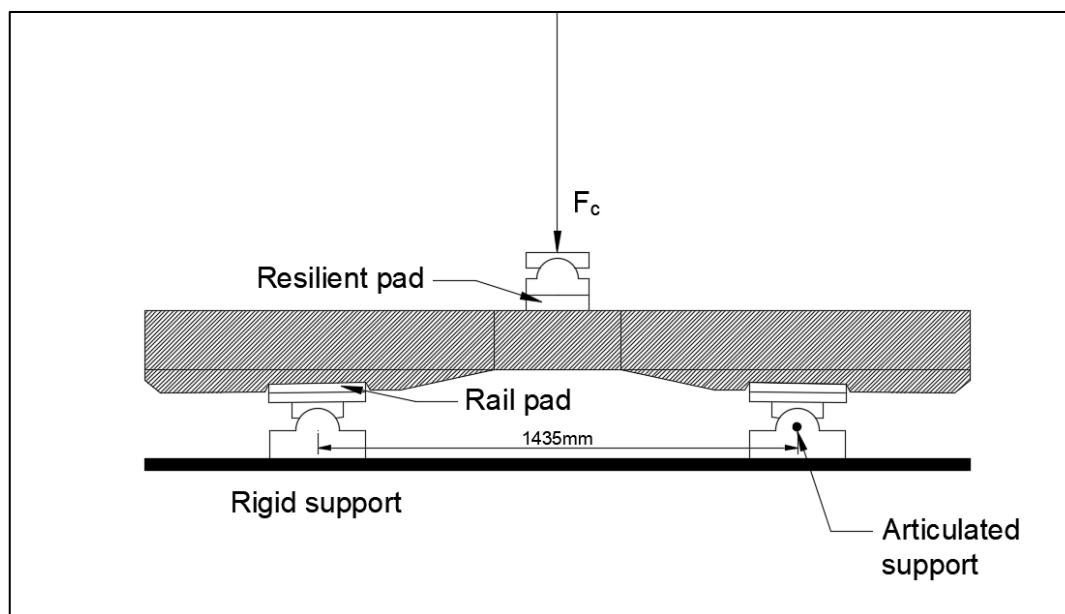
Parts	Modulus of elasticity (MPa)	Density (kg/m ³)	Poisson's ratio
Sleeper	33000	2400	0.23
Ballast	1500	1800	0.20
Tendons	200000	9800	0.30

230

Table 3. Material properties used in the FE model

231 5.1 Model validation

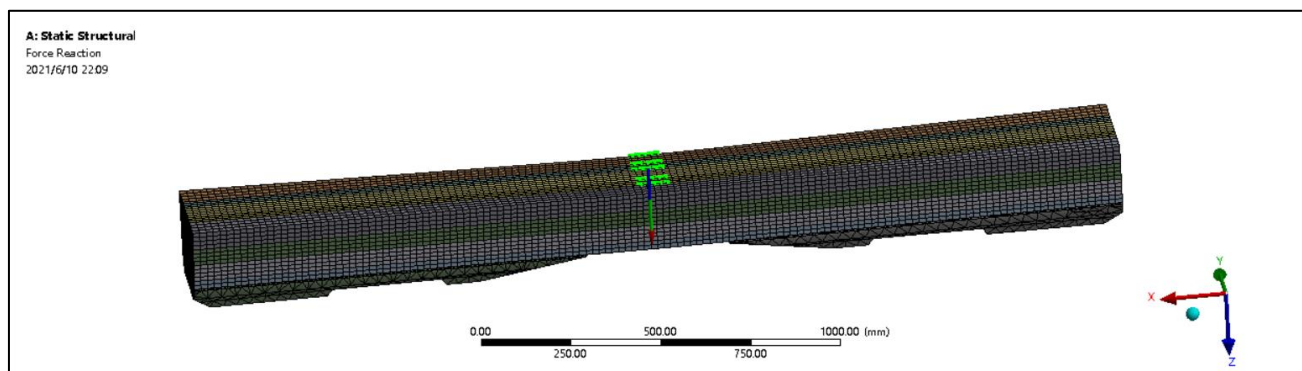
232 To validate the model, the static capacity test of Chinese Type III prestressed concrete sleeper is used
 233 to verify structural and material properties of the FE sleeper model [23, 24]. An experimental
 234 programme as conducted at Beijing Jiaotong University (apparatus shown in **Figure 5**) used to validate
 235 the FE sleeper model. In the experiment, the loading jack was applied to a rubber plate located just
 236 above the surface of the centre of the sleeper. The loading level increases up to 140kN. The DIC (digital
 237 image correlation) was selected to obtain load-deflection response.



238

239

Figure 5. Apparatus of the static capacity test of sleeper

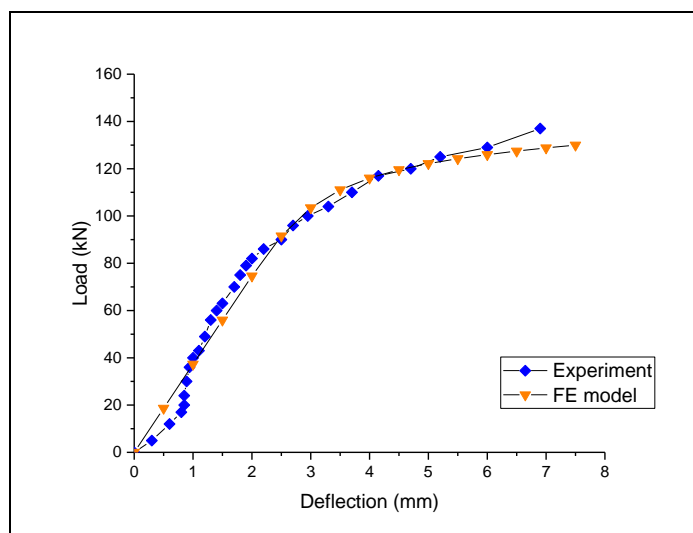


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Figure 6. Static capacity test simulation for FE sleeper model validation

242 In order to validate the quality of the FE sleeper model, the numerical results have been compared with
 243 experimental results from the static capacity test in previous research [26]. **Figure 6** shows the static
 244 capacity simulation for FE sleeper model material and structural validation. **Figure 7** presents the
 245 comparison between the FE analysis and experimental results. The results are found to be in very good
 246 agreement with the static capacity test. The maximum difference between the experimental and
 247 numerical results is only 5.99%.



248

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Figure 7. Load-deflection response between FE results and experimental results

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6 Results of the fatigue life prediction

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Section 2 introduces the theoretical fatigue life assessment method. The cyclic loads (dynamic loads) between 55kN to 365kN are chosen to calculate the fatigue life. Each dynamic load is assumed as constant which can calculate only one fatigue life for both numerical and theoretical assessment method. The output results represent the sleeper will fail under the constant dynamic load after output number of cycles. The simulation of fatigue life refers to the results of theoretical fatigue life assessment. **Figure 8** presents the cyclic loads setting. The pressure is applied at the rail seat area and the magnitude range of cyclic loads (dynamic loads) is following theoretical calculation in order to compare the results. The S-N curve used in predicting fatigue life is shown in **Table 1** of Section 2. The comparison of numerical and theoretical fatigue life results is demonstrated in **Table 4** and **Figure 9**. From **Table 4**, it can be seen that the errors between numerical and theoretical results range from 0.03% to 35.13%. The average error is 13.03%. From **Figure 9**, it is seen that the FE analysis results are quite similar to the theoretical fatigue life results. In order to validate the numerical fatigue model, the experimental data is also used to compare with numerical and theoretical results. Parvez and Foster have conducted fatigue experiments of prestressed concrete sleepers to observe failure cycles [25, 26]. Two specimens are chosen from fatigue experiments to calculate the average of the failure cycles presented in **Table 5**. Based on their test results, the comparison between experimental, theoretical, and numerical results is shown in **Table 6**. A good correlation between the numerical, theoretical, and experimental results provides a reliable method for further parametric study.

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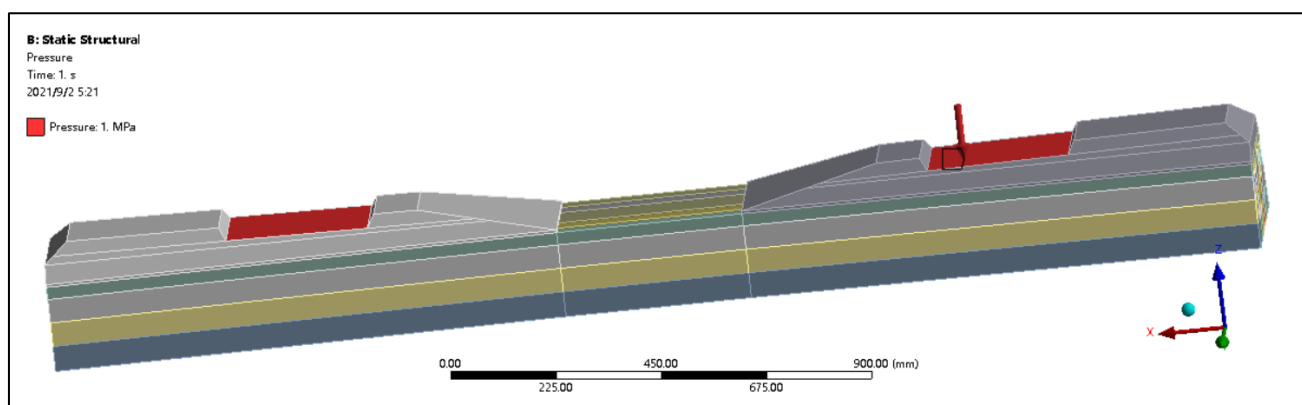
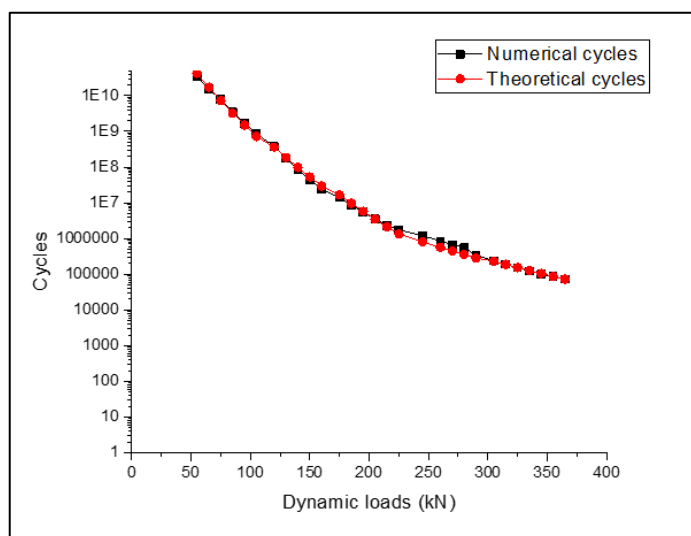


Figure 8. Setup for fatigue life simulation

Load (kN)	Life cycles (numerical)	Life cycles (theoretical)	Error %
55	3.47E+10	4.19E+10	20.59
65	1.54E+10	1.70E+10	10.31
75	7.71E+09	7.24E+09	6.19
85	3.43E+09	3.23E+09	5.95
95	1.68E+09	1.50E+09	10.80
105	8.79E+08	7.21E+08	17.94
120	3.76E+08	3.59E+08	4.39
130	1.73E+08	1.85E+08	6.70
140	8.48E+07	9.77E+08	15.14
150	4.38E+07	5.30E+07	20.95
160	2.39E+07	2.95E+07	23.41
175	1.41E+07	1.68E+07	18.75
185	8.62E+06	9.74E+06	13.03
195	5.41E+06	5.77E+06	6.66
205	3.48E+06	3.48E+06	0.03
215	2.30E+06	2.14E+06	6.92
225	1.76E+06	1.33E+06	24.21
245	1.21E+06	8.04E+05	33.36
260	8.49E+05	5.61E+05	33.89
270	6.79E+05	4.45E+05	34.45
280	5.48E+05	3.56E+05	35.13
290	3.44E+05	2.86E+05	16.92
305	2.34E+05	2.31E+05	1.30
315	1.88E+05	1.88E+05	0.09
325	1.52E+05	1.54E+05	1.00
335	1.23E+05	1.26E+05	2.76
345	1.00E+05	1.05E+05	4.37
355	85186	86773	1.86
365	72829	72364	0.64

Table 4. Comparison of theoretical and numerical fatigue life results



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273

Figure 9. Comparison between numerical and theoretical fatigue life results

Specimen ID	Failure cycles	Average	Standard Deviation
SF2-a	773793	896290	173236
SF3-a	1018787		

274

Table 5. Experimental failure cycles [25]

Failure cycles				
Experimental result	Theoretical result	Deviation ratio %	Numerical result	Deviation ratio %
896290	889577	0.75	849000	5.28

275

Table 6. Comparison between experimental, theoretical, and numerical results

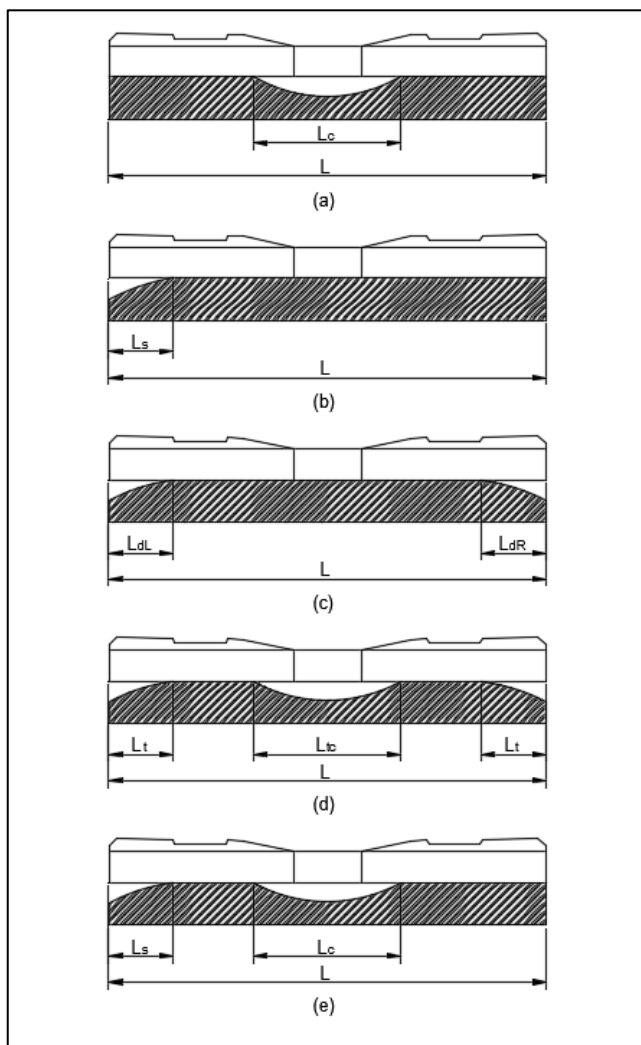
276 7 Parametric study

277 The comparison between the theoretical and model results implies that the fatigue life under various
 278 dynamic loads can be predicted and simulated use the FE model. Therefore, the validated FE model
 279 can be utilised to analyse the fatigue life under other critical conditions.

280 7.1 Support conditions

281 In a conventional track system, railway sleepers are usually laid on ballast and subgrade. It is usually
 282 assumed there is full contact between sleeper and ballast for analysis and design purposes. However,
 283 voids and hanging support conditions could occur and can cause problems to both sleeper and track
 284 system. Previous studies [27, 28] indicate 5 typical sleeper/ballast contact patterns used to analyse how
 285 fatigue life is affected by the support conditions. The sleeper/ballast contact patterns are illustrated in
 286 **Figure 10.** Full contact between sleepers and ballast is typically assumed for analysis and design

287 purposes. The voids and pockets in the sleeper/ballast contact interface form between sleepers and the
 288 ballast underneath that could cause problems to both the sleepers and the track system as a whole. The
 289 selected support patterns are practical concerns in actual railway track problems. In the simulation, the
 290 positive reaction at rail-seat and the negative reaction at mid-span are considered and the fatigue life
 291 depends on the minimum of output. The ratio of the void length and sleeper length is controlled to
 292 observe the change of fatigue life under different support patterns. The simulation has been conducted
 293 for each pattern. Loads of between 55kN to 325kN are applied in the FE model.



294

295 **Figure 10.** Sleeper/ballast contact patterns: (a) central void, (b) single hanging, (c) double hanging, (d)
 296 triple hanging, (e) side-central voids

297

298 7.1.1 Central void

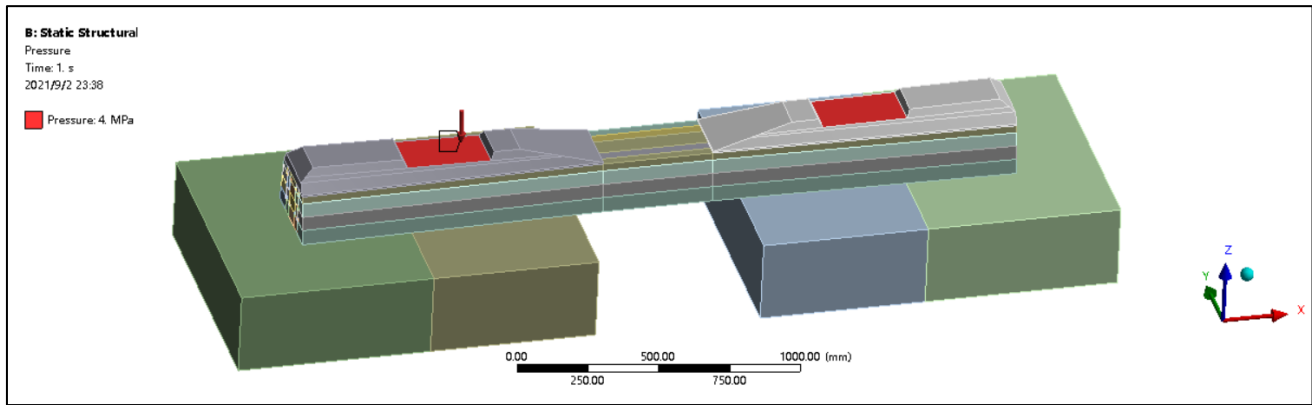
299 The central void is a void which forms at the centre of the sleeper and expands symmetrically in both
 300 directions. The ratio of the central void length to the sleeper length is given:

$$301 \quad \alpha_c = \frac{L_c}{L} \quad (11)$$

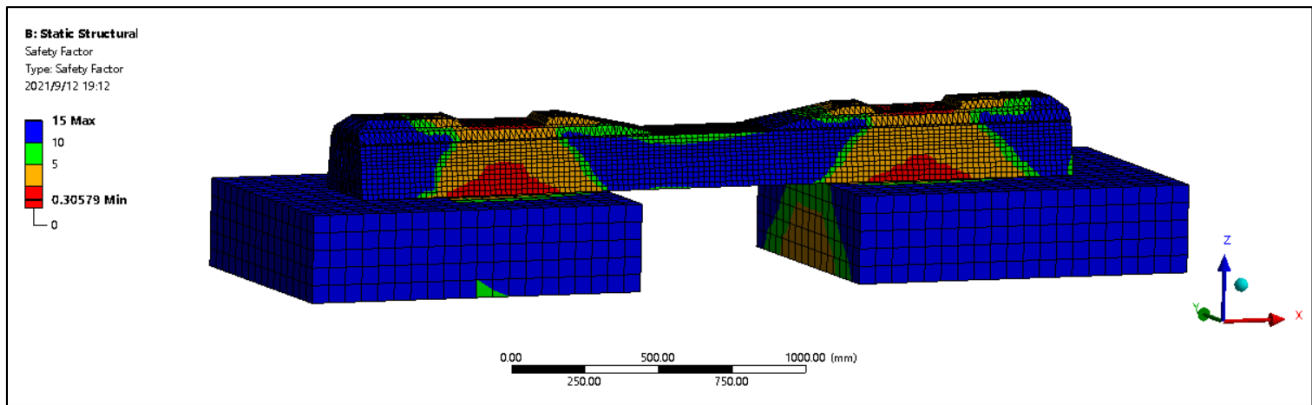
302 where L_c is the central void length; L is the sleeper length.

303 The ratio α_c of 30.77% (800/2600), 38.46% (1000/2600), 46.15% (1200/2600), 53.85% (1400/2600)
 304 were respectively simulated to analyse fatigue life. **Figure 11** shows the simulation setup and stress
 305 distribution contour. In the FE model, the length of central void area can be adjusted and expanded in
 306 both directions. **Figure 12** indicates the results of fatigue life in the central void support condition.

307



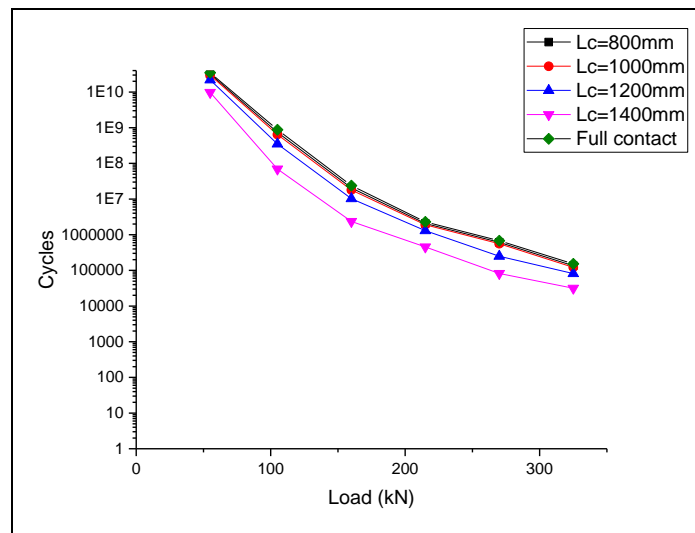
308



309

Figure 11. Simulation of the central void support condition

310



311

Figure 12. Comparison of the central void pattern results

312 **Figure 11** shows the maximum stress occurs at bottom rail-seat area, which the minimum fatigue life
 313 should happen at rail-seat area. The effect of the central void on the fatigue life of the prestressed
 314 concrete sleeper is shown in **Figure 12**. The results show that central void length of less than 1000mm
 315 (ratio α_c less than 38.46%) is unlikely to influence the fatigue life in comparison with full contact
 316 pattern, while the change of fatigue life is just around 2%. The central void length increases to 1200mm
 317 (ratio $\alpha_c=46.15\%$), and the fatigue life falls by 51% on average for loads of between 55kN to 325kN.
 318 When the central void length increases to 1400mm (ratio $\alpha_c=53.85\%$), the fatigue life reduces by 83%.
 319 It is obvious that the fatigue life is insensitive when the small void ratios from 0 to 40% are used.

320 7.1.2 Single hanging

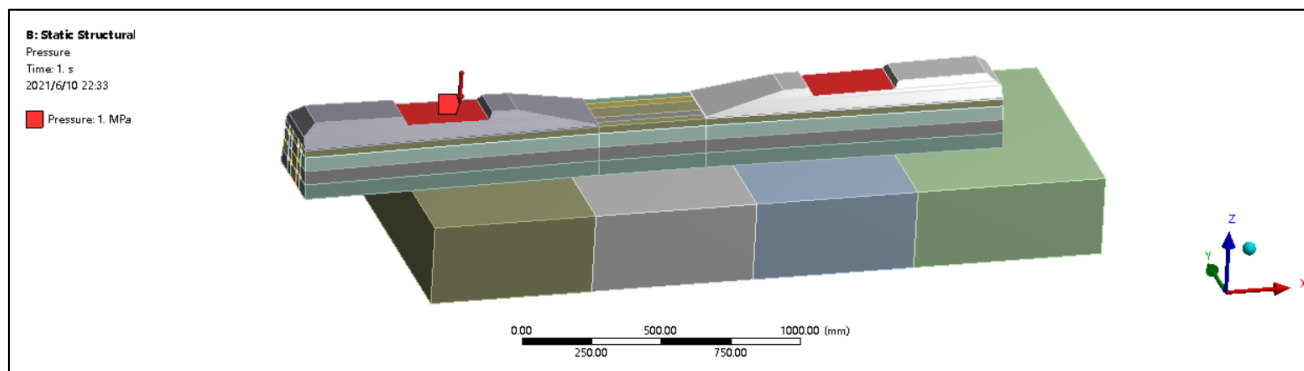
321 The single hanging is a void that forms at the one end of the sleeper and grows incrementally to the
 322 other end of the sleeper. The ratio of the single side void length to the sleeper length is given as:

$$323 \alpha_s = \frac{L_s}{L} \quad (12)$$

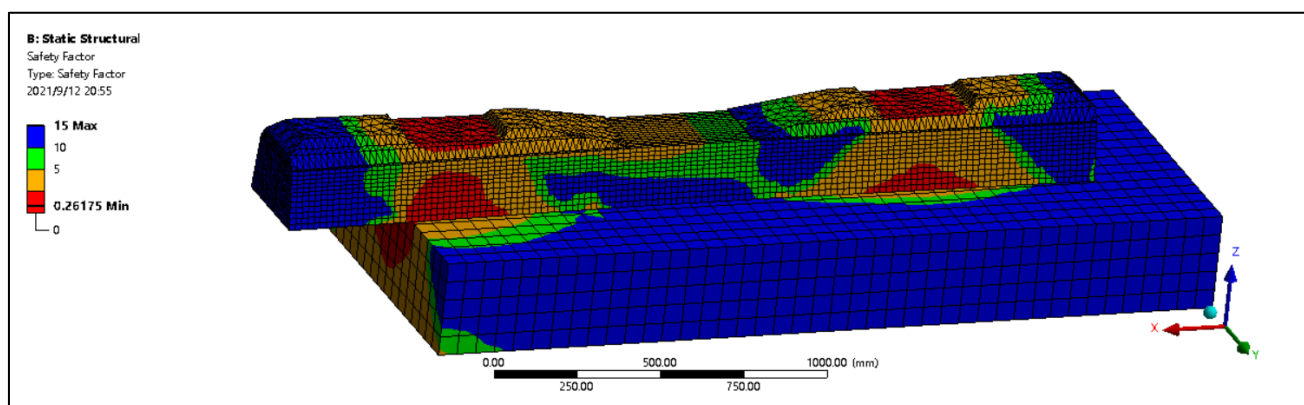
324 where L_s is the side void length; L is the sleeper length.

325 The ratio α_s of 11.54% (300/2600), 15.38% (400/2600), 19.23% (500/2600), 23.08% (600/2600)
 326 were respectively simulated to analyse the fatigue life. **Figure 13** shows the simulation setup and stress
 327 distribution contour. In the FE model, the length of the single side void area can be adjusted and
 328 expanded in the other direction. **Figure 14** indicates the results of fatigue life in the single hanging
 329 support condition.

330

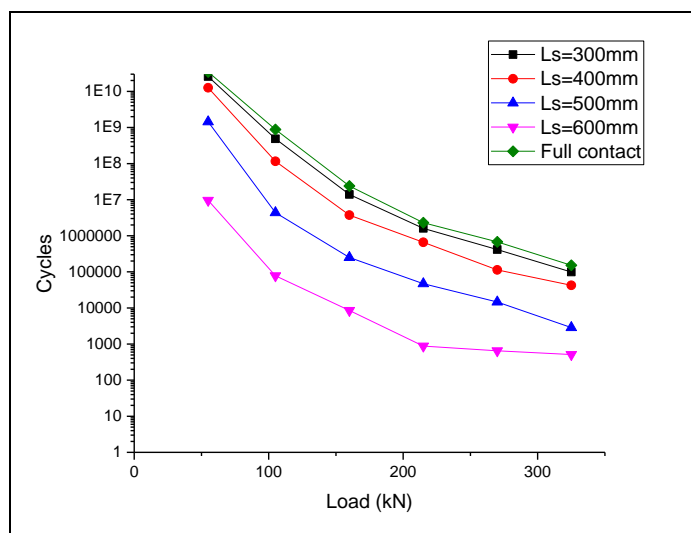


331



332

Figure 13. Simulation of the single hanging support condition



333

334

Figure 14. Comparison of the single hanging pattern results

335 **Figure 13** shows the maximum stress occurs at bottom rail-seat area, which the minimum fatigue life
 336 should happen at rail-seat area. **Figure 14** presents the effect of the one end hanging condition on the
 337 fatigue life of the prestressed concrete sleeper. The fatigue life seems to be significantly affected by
 338 this support condition. Single hanging void lengths ranging from 300mm to 600mm are investigated.
 339 The fatigue life decreases when single hanging void length increases. When the side hanging length is
 340 300mm (ratio $\alpha_s=11.54\%$), the fatigue life reduces by 35.98% in average. The difference between
 341 400mm (ratio $\alpha_s=15.38\%$), 500mm (ratio $\alpha_s=19.23\%$) voids and the full contact pattern are 76.89%
 342 and 98.04% respectively. When the side hanging length increases to 600mm (ratio $\alpha_s=23.08\%$), the
 343 fatigue life reaches half of the full contact support pattern which is a 99.90% change.

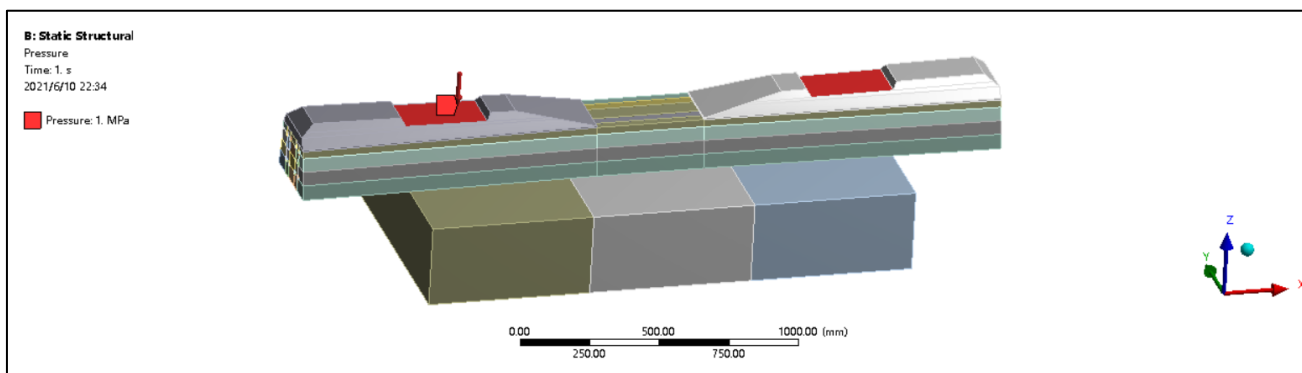
344 7.1.3 Double hanging

345 The double hanging is when voids are present at both ends of the sleeper. The ratio of the double side
 346 void length to the sleeper length is given as:

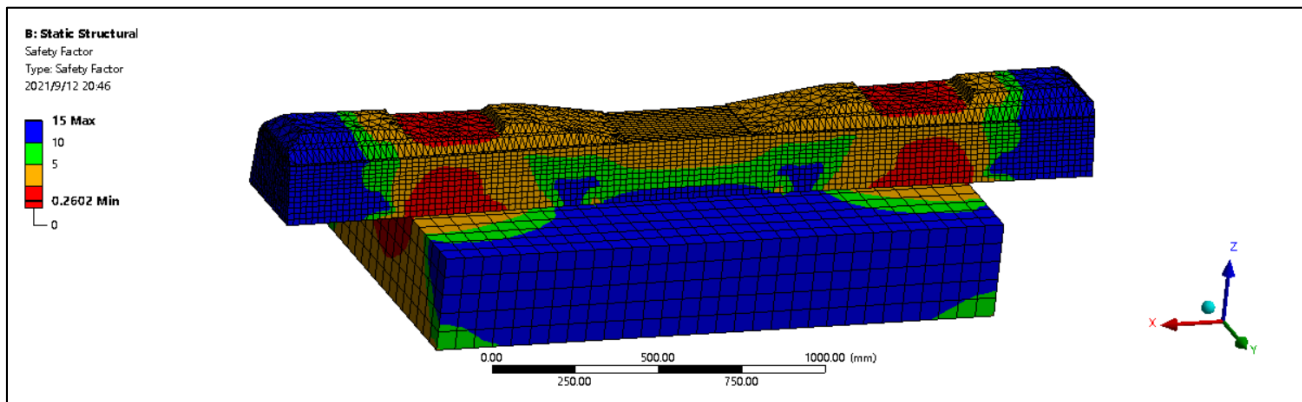
$$347 \quad \alpha_d = \frac{L_{dL}}{L}, \beta_d = \frac{L_{dR}}{L} \quad (13)$$

348 where L_{dR}, L_{dL} are the void length at right and left sides; L is the sleeper length.

349 The ratio α_d, β_d of 11.54% (300/2600), 15.38% (400/2600), 19.23% (500/2600), 23.08% (600/2600)
 350 are respectively simulated to analyse the fatigue life. **Figure 15** shows the simulation setup and stress
 351 distribution contour. In the FE model, the length of each side void area can be adjusted and expanded
 352 in each direction. The void length on both sides is assumed equal. **Figure 16** indicates the results of
 353 fatigue life in the double hanging support condition.



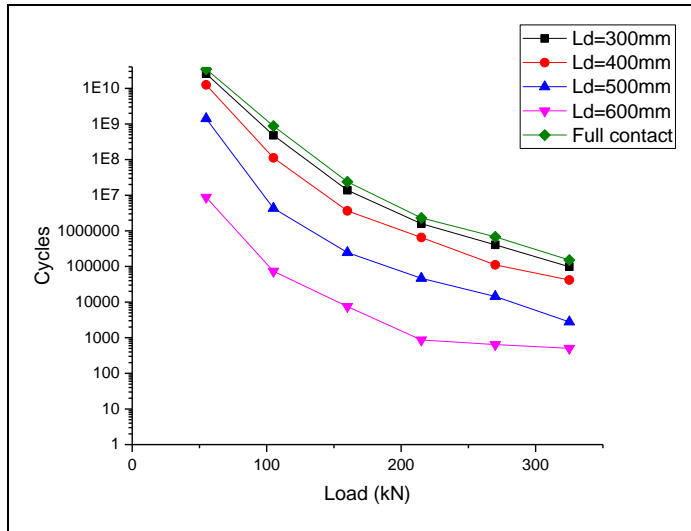
354



355

356

Figure 15. Simulation of the double hanging support condition



357

358

Figure 16. Comparison of the double hanging pattern results

359 **Figure 15** shows the maximum stress occurs at bottom rail-seat area, which the minimum fatigue life
 360 should happen at rail-seat area. The changes in fatigue life of the prestressed concrete sleeper for
 361 varying lengths of double hanging voids are illustrated in **Figure 16**. The results are similar to the
 362 single hanging support pattern. Single hanging void lengths from 300mm to 600mm are investigated.
 363 The fatigue life decreases as single hanging void length increases. With the side hanging length at
 364 300mm (ratio $\alpha_d=11.54\%$), the fatigue life reduces by 36.71% on average. When the side hanging

365 length increases to 600mm (ratio $\alpha_d=23.08\%$), the fatigue life only reaches half of the full contact
 366 support pattern.

367 7.1.4 Triple hanging

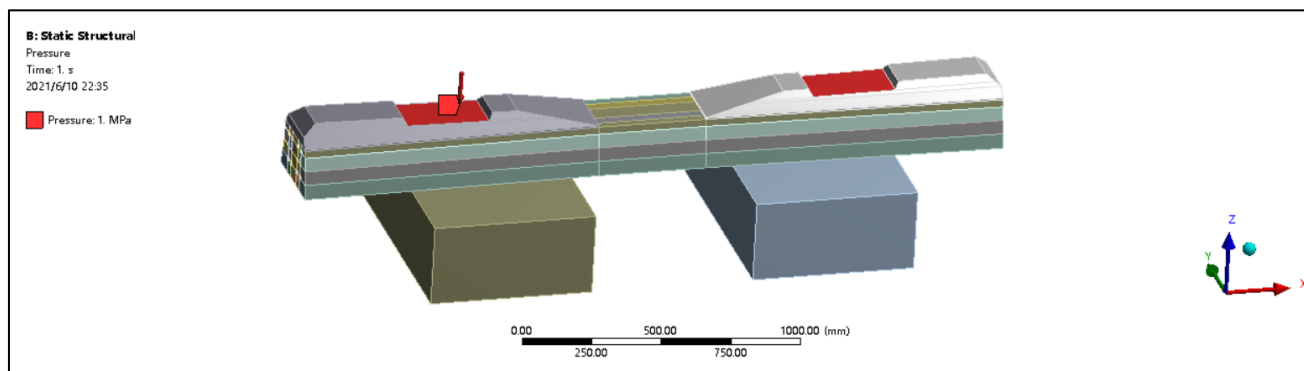
368 The triple hanging is when there are voids at both ends and a pocket in the middle of the sleepers. The
 369 ratio of the triple side void length to the sleeper length is given as:

$$370 \quad \alpha_t = \frac{L_t}{L}, \beta_t = \frac{L_{tc}}{L} \quad (14)$$

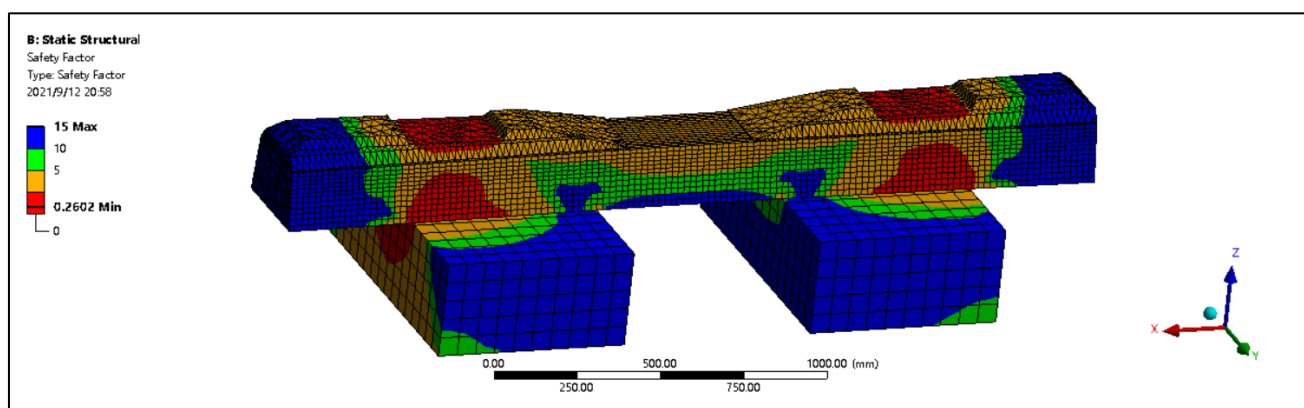
371 where L_t is the void length at sides; L_{tc} is the void length at the centre; L is the sleeper length.

372 The ratio α_t, β_t of 11.54%/30.77% (300/2600; 800/2600), 19.23%/38.46% (500/2600; 1000/2600),
 373 23.08%/30.77% (600/2600; 800/2600), 23.08%/46.15% (600/2600; 1200/2600) are respectively
 374 simulated to analyse the fatigue life. **Figure 17** shows the simulation setup and stress distribution
 375 contour. In the FE model, the length of each end void and centre void area can be adjusted and expanded.
 376 Only symmetrical cases are considered in this research. **Figure 18** indicates the results of fatigue life
 377 in the triple hanging support condition.

378

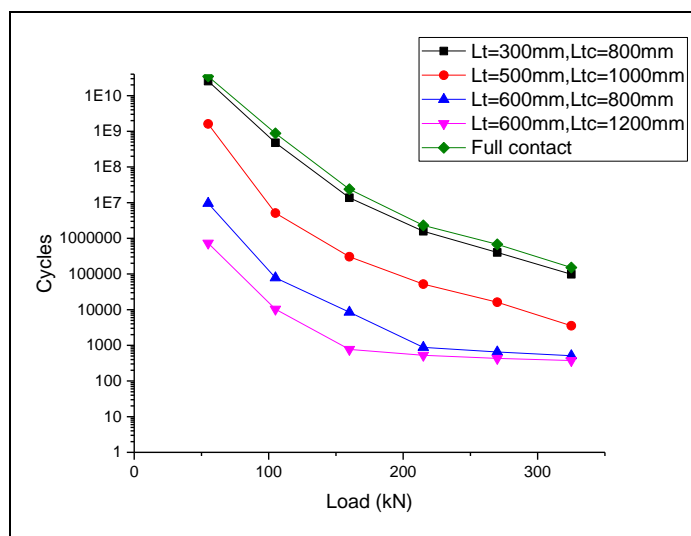


379



380

Figure 17. Simulation of triple hanging support condition



381

382

Figure 18. Comparison of the triple hanging pattern results

383 **Figure 17** shows the maximum stress occurs at bottom rail-seat area, which the minimum fatigue life
 384 should happen at rail-seat area. The fatigue life of the voided prestressed concrete sleeper under triple
 385 void contact pattern is demonstrated in **Figure 18**. It is found that a small void ratio (α_t , β_t of
 386 11.54%/30.77% (300mm/2600mm; 800mm/2600mm)) does not influence the fatigue life very much.
 387 In comparison with α_t , β_t of 19.23%/38.46% (500mm/2600mm; 1000mm/2600mm) and α_t , β_t of
 388 23.08%/30.77% (600mm/2600mm; 800mm/2600mm), side voids have more sensitive effects than
 389 central voids. To observe the changes in large side hanging with different central voids, the α_t , β_t of
 390 23.08%/30.77% (600mm/2600mm; 800mm/2600mm) and 23.08%/46.15% (600mm/2600mm;
 391 1200mm/2600mm) are investigated. At loads of between 55kN and 160kN, the large central voids of
 392 triple hanging support pattern have less fatigue life. However, when the large loads are applied (more
 393 than 215kN), the fatigue life becomes very similar to each other. This demonstrates that poor support
 394 conditions under large dynamic loads can easily result in failure on the prestressed concrete sleepers.

395 7.1.5 Side-central voids

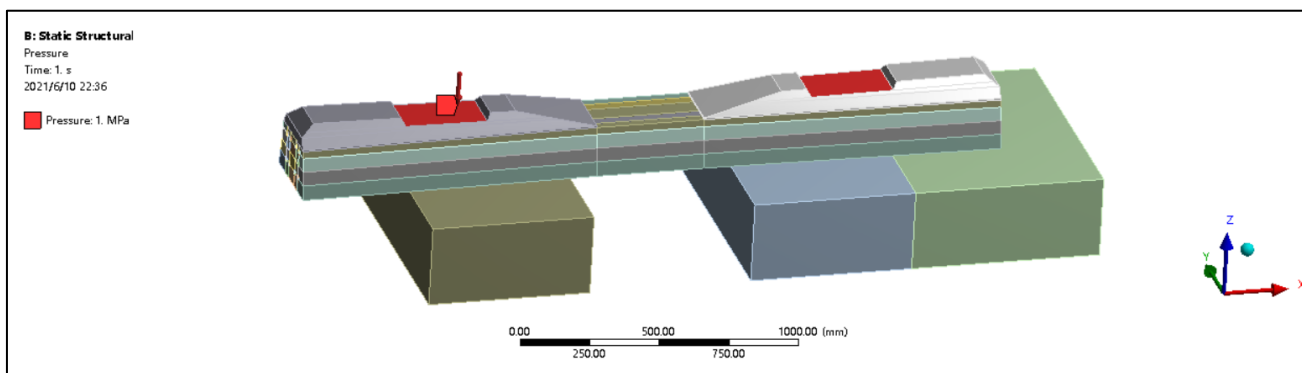
396 The side-central voids are formed at the edge hanging and central void at the same time. The ratio of
 397 the side-central voids length to the sleeper length is given as:

$$398 \alpha_{s-c} = \frac{L_s}{L}, \beta_{s-c} = \frac{L_c}{L} \quad (15)$$

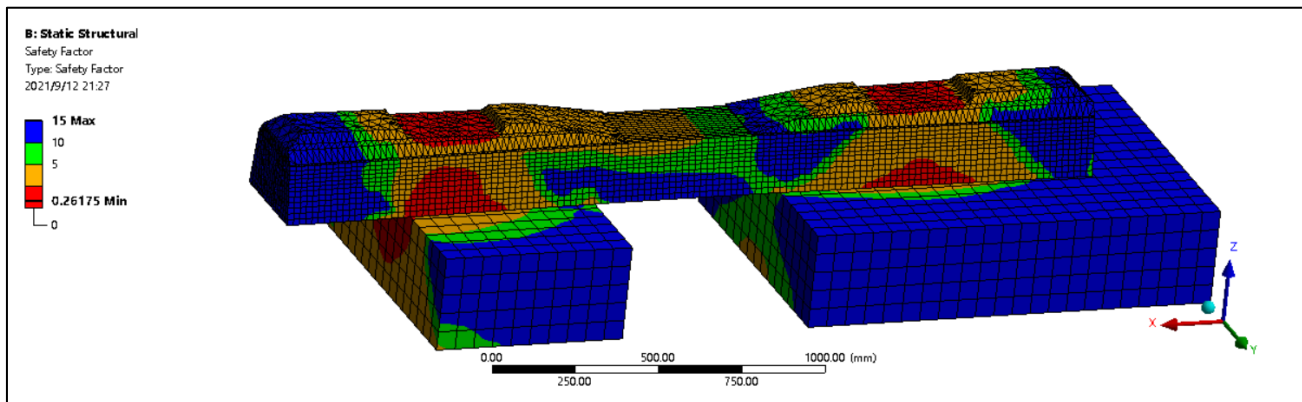
399 where L_{dR} , L_{dL} are the void length at right and left sides; L is the sleeper length.

400 The ratio α_{s-c} , β_{s-c} of 11.54%/30.77% (300/2600; 800/2600), 19.23%/38.46% (500/2600;
 401 1000/2600), 23.08%/30.77% (600/2600; 800/2600), 23.08%/46.15% (600/2600; 1200/2600) are
 402 respectively simulated to analyse the fatigue life. **Figure 19** shows the simulation setup and stress
 403 distribution contour. In the FE model, the length of edge and central void area can be adjusted and
 404 expanded. The void length at both sides is assumed equal. **Figure 20** indicates the results of fatigue
 405 life in the side-central void support condition.

406



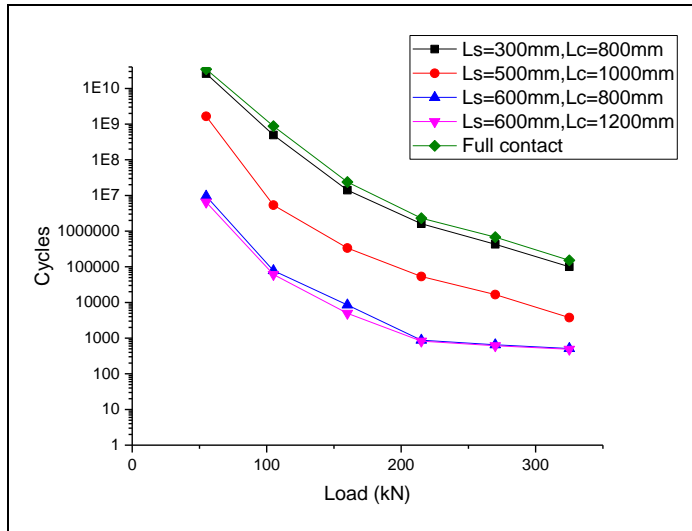
407



408

Figure 19. Simulation of side-central void support condition

409



410

Figure 20. Comparison of the side-central void pattern results

411 **Figure 19** shows the maximum stress occurs at bottom rail-seat area, which the minimum fatigue life
 412 should happen at rail-seat area. The changes in the fatigue life of the prestressed concrete sleepers due
 413 to side and central voids are illustrated in **Figure 20**. This contact pattern can be considered to be a
 414 combination of central void and single hanging. In this support pattern, small voids (α_{s-c} , β_{s-c} of
 415 11.54%/30.77% (300mm/2600mm; 800mm/2600mm)) do not have much influence on the fatigue life
 416 with the change being just 35.14%. From **Figure 20**, it can be seen that the same side voids but different
 417 central voids (α_t , β_t of 23.08%/30.77% (600mm/2600mm; 800mm/2600mm) and 23.08%/46.15%

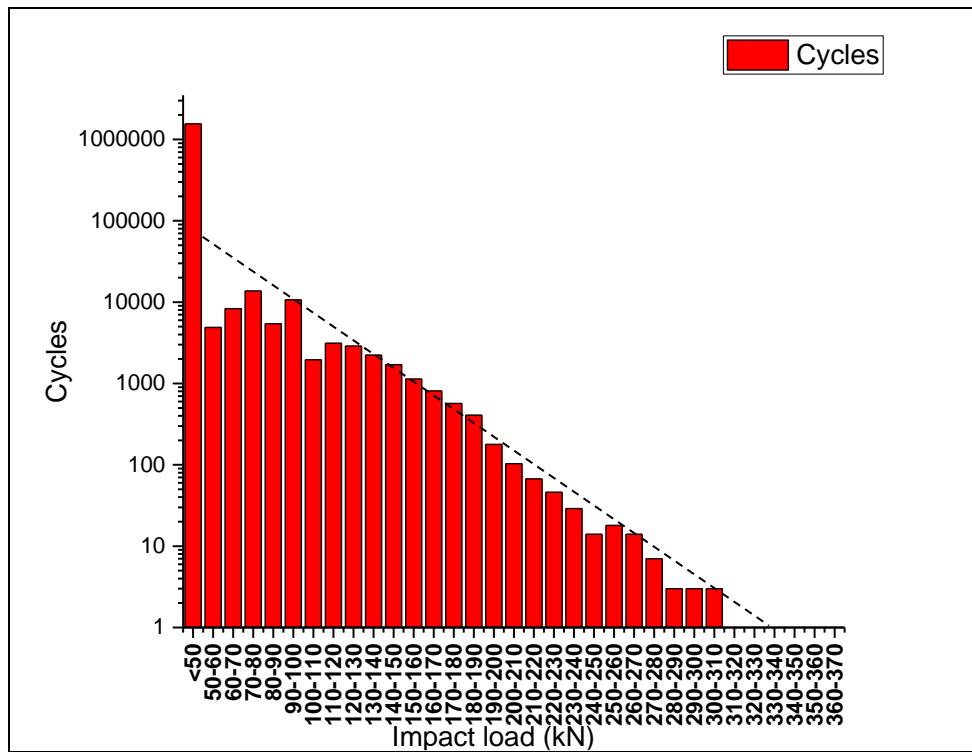
418 (600mm/2600mm; 1200mm/2600mm)), the results of the fatigue lives are very similar. The results
419 also demonstrate the side voids have more influence than central voids.

420 7.2 *Impact load distribution*

421 The FE model analyses the fatigue lives under various magnitude of dynamic loads. The output results
422 of fatigue lives are under a constant dynamic load. In the field, the railway sleepers could experience
423 a wide range of dynamic loads. This section analyses the fatigue life under different impact load
424 distributions. In this section, the support condition is set as full contact pattern.

425 A comprehensive investigation of actual impact loads was conducted by Leong, J and Kaewunruen, S
426 (2007, 2007) [7, 28]. The frequency of occurrence of impact loads per year has been recorded, as shown
427 in **Figure 21**. Over 1 year, data were obtained from two Teknis Wheel Condition Monitoring stations
428 located on different heavy-haul mineral lines in Australia. A total of nearly six million passing wheels
429 were measured. A number of 1,609,712 passing wheels measured for a year from unit trains with 26
430 tons to 28 tons axle loads were used in fatigue life analysis. The analysis of field measurement indicates
431 over 96% of wheels created impact loads less than 50kN. However, there were still more than 40,000
432 passing wheels higher than 140kN. From **Table 4** above, it can be seen that large loads can significantly
433 cause a reduction of fatigue life. Therefore, the effect of impact load distribution needs to be studied.
434 The impact load distribution data is applied in the FE model. MATLAB is used to generate an impact
435 load distribution and inputted into the numerical model.

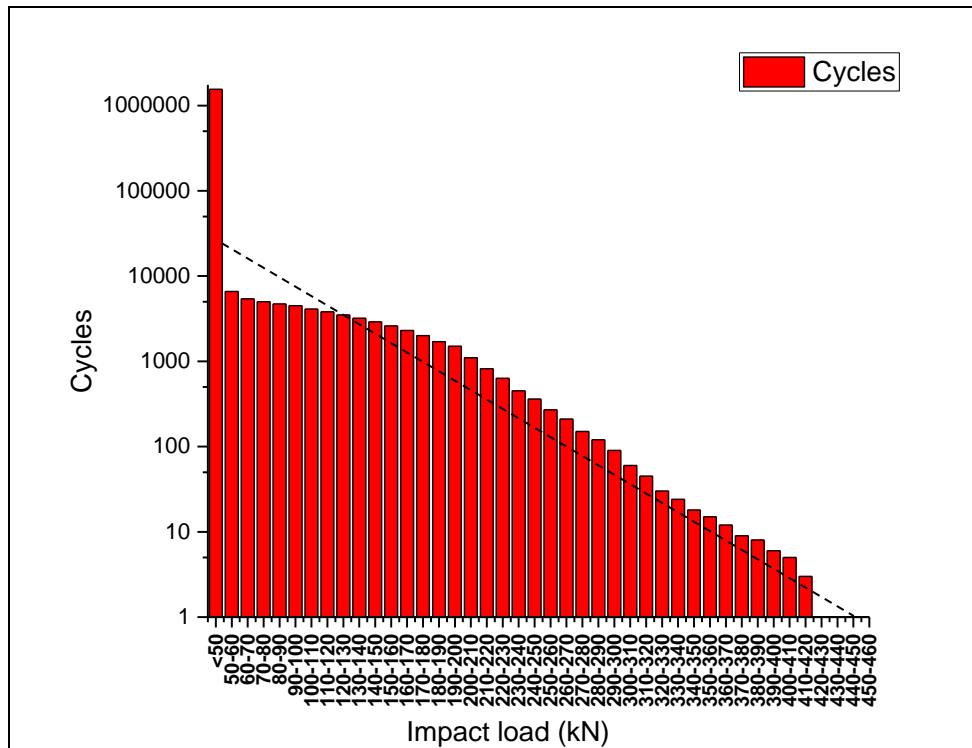
436 The dashed line shown in **Figure 21** demonstrates the impact load distribution. The slope of the dashed
437 line controls the impact load magnitude range and volume of occurrence of impact loads (the total
438 cycles need to be close to field measurement). In this section, two cases of the impact load distribution
439 are investigated and the field measurement data is regarded as the control group. Case 1 is the range of
440 impact load up to 410kN with low volume of occurrence. Case 2 shows the impact load range up to
441 240kN but low volume of occurrence. The impact load distribution of Case 1 and Case 2 are shown in
442 **Figure 22** and **Table 7**.



443

444

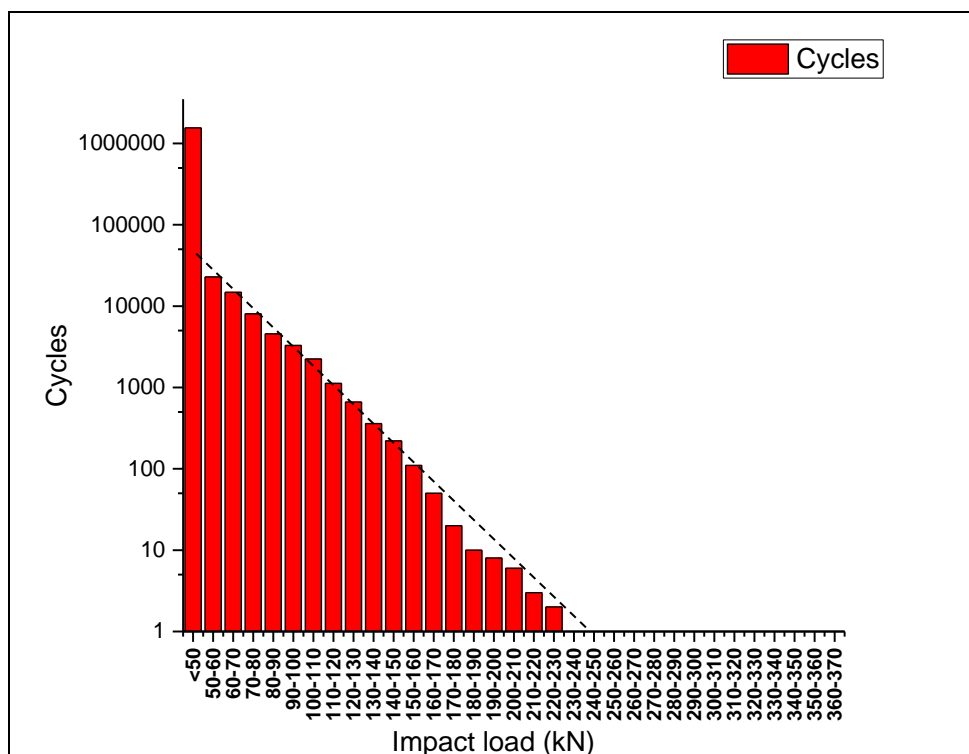
Figure 21. Field measurement of impact loads per year



445

446

(a) Case 1



(b) Case 2

Figure 22. Case study of impact load distributions

	<50kN	50-70kN	70-140kN	140-210kN	210kN<	Total cycles
Case 1	1551481	12000	28800	14920	2484	1609711
Case 2	1551481	37600	20240	427	3	1609751
Field data	1551481	13179	39951	4963	138	1609712

Table 7. Impact load distributions

450

451 From **Table 10**, it can be seen that the total passing wheels per year for Case 1 and Case 2 are 1,609,711
 452 and 1,609,751 respectively. They are very close to field measurement cycles (1,609,751 cycles) to
 453 ensure the distributions can be compared reasonable. Small impact loads (less than 50kN) for both
 454 Case 1 and Case 2 follow field data because small impact loads don't have significant effect on the
 455 fatigue life.

456 The total cycles (passing wheels) per year for Case 1, Case 2, and Field data are assumed following
 457 **Table 7** and keep the same every year, the damage index in each year can be determined. According
 458 to Miner's rule (Equation (1)), the fatigue life in years can be calculated. The outcome of the fatigue
 459 life with impact load distributions is shown in **Table 8**. There is not much difference in fatigue life
 460 between Case 1, Case 2, and the field measurement data as 96% of total cycles are small impact loads.
 461 From **Table 8**, it can be seen that the result of Case 1 is 14.28% less than field measurement data
 462 whereas the result of Case 2 is 8.51% higher than field measurement data. The results demonstrate
 463 greater large impact load occurrence can lead to the reduction of the service life of prestressed concrete
 464 sleepers. **Table 4** shows that the fatigue life is up to $4.19E+10$ cycles under constant 55kN dynamic
 465 load which also indicates the large impact loads need to be controlled for increasing the service life of
 466 railway sleepers.

Impact load distribution	Life cycles	Years
Case 1	8.57E+08	32
Case 2	1.09E+09	37
Field data	1.00E+09	35

467 **Table 8.** Results of the fatigue life under different impact load distributions

468 7.3 Track stiffness

469 Track stiffness can influence the dynamic behaviour of trains, bearing capacity, track geometry, and
 470 service life of sleepers. This section discusses the effect of track stiffness on the fatigue life of
 471 prestressed concrete sleepers. In this section, the support condition is set as full contact pattern.

472 Previous research conducted by Martin, X et al (2010) [29] has investigated three levels of track
 473 stiffness ('soft', 'normal', and 'stiff') according to the static approach. The track properties are shown
 474 in **Table 9**. In their research, 100kN wheel load is applied. These track stiffness properties are input
 475 into the FE model to analyse the performance of the railway sleeper. The simulation results for 'soft',
 476 'normal', and 'stiff' track are presented in **Table 10**.

	Track A: 'soft'	Track B: 'normal'	Track C: 'stiff'
Stiffness of ballast/subgrade	10kN/mm	50kN/mm	100kN/mm
Rail displacement	3.16mm	1.28mm	0.58mm

477 **Table 9.** Track stiffness properties

Track type	Life cycles	Years
Soft	1.42E+09	42
Normal	1.08E+09	37
Stiff	9.63E+08	34

478 **Table 10.** Results of the fatigue life in different track stiffness

479 The results from **Table 13** indicate stiffer track have shorter life cycles. In comparison with these three
 480 types of tracks, the fatigue life of 'soft' track is 31.54% more than 'normal' track while that of 'stiff'
 481 track is 10.69% less than 'normal' track. The results show that high track stiffness increases dynamic
 482 load in the wheel-rail reaction and causes the reduction of life cycles. However, these results do not
 483 indicate stiffer tracks have worse performance than softer tracks because the parametric study only
 484 considers how track stiffness influences fatigue life. In actual track systems, relatively high track
 485 stiffness can provide sufficient track resistance and reduce deflections.

486 8 Conclusion

487 This paper aimed to develop a numerical model of prestressed concrete sleepers to effectively simulate
 488 the fatigue life and performance of railway sleepers under cyclic loads. The objective was to better
 489 understand mechanisms of fatigue problems of prestressed concrete sleepers by using an analytical
 490 model. To investigate fatigue life, the stress life method was chosen to develop a numerical model.
 491 This method is based on an empirical S-N curve and modified by a variety of factors. The theoretical
 492 assessment method is also presented in order to validate the numerical model. The model predictions
 493 for fatigue life successfully matched with theoretical results. Based on the obtained results of this study,
 494 the key findings are revealed as follows:

- 495 • Generally, dynamic loads significantly affect fatigue life. Fatigue life of prestressed concrete
496 sleepers is inversely proportional to the magnitude of dynamic loads. This phenomenon indicates
497 the high impact loads need to be controlled.
- 498 • Based on Stress Life method of fatigue analysis type, a numerical analysis of fatigue life has been
499 conducted using FEM. The previous theoretical fatigue life assessment method is used for
500 validation of the numerical fatigue life model.
- 501 • In numerical study, the S-N curve is used to define the alternating stress for calculating fatigue life.
502 The equivalent alternating stress is determined by empirical stress theory.
- 503 • Statistical analysis of the fatigue loads from field measurement and their fatigue performance on
504 the prestressed concrete sleeper.
- 505 • Development of reasonable numerical fatigue life model can be used on prestressed concrete
506 sleepers. However, in practice, dynamic loads could be various due to different contact conditions.
507 Therefore, the results of fatigue life only considering constant load are the limit to this research.
- 508 • On this point, it is important to note that rail/wheel irregularity needs to be considered in further
509 study.

510 Parametric studies investigate the support conditions, impact load distribution, and track stiffness,
511 which influence the fatigue life of railway sleeper. In general, the railway sleepers in conventional
512 tracks are considered to have an ideal contact between the sleeper and ballast. However, in some
513 situations problems could result in voids and pockets formed in the sleeper/ballast contact. Five support
514 conditions are investigated. The stress distributions of the sleepers are analysed in order to determine
515 the minimum fatigue life of each support patterns. In the support condition study, the positive moment
516 at rail-seat is more critical than negative moment at midspan, which means the minimum fatigue life
517 usually occurs at bottom rail-seat area. Central void support patterns do not influence the fatigue life
518 much. In addition, regular tamping activities typically concentrate the ballast reaction below the rail
519 seats, which is similar to central void support pattern. When the voids occur at the side of prestressed
520 concrete sleepers, the life cycles reduce. With the side voids increasing, the life cycles decrease sharply.
521 In actual life, the full support pattern is favorable for the rail seat sections but could be unfavorable for
522 the centre section. Therefore, it is necessary to conduct track inspection to ensure well supported
523 conditions. The results of impact load distributions indicate a greater proportion of high impact loads
524 affect service life. Small proportions (1.1%) of high impact loads can cause 14% life cycle reduction.
525 If the dynamic loads can be controlled no more than 70kN, the performance of railway sleepers is
526 improved. The investigation of track stiffness reveals stiffer track can decrease the life cycles. High
527 track stiffness results in increases of dynamic loads in wheel-rail reaction as well as sleepers, which
528 causes the reduction of life cycles. It should be noted that the experimental programs are suggested to
529 conduct in future research, which can be compared with theoretical and numerical results for more
530 accurate prediction.

531 This article demonstrates a reliable approach for evaluating the fatigue life of prestressed concrete
532 sleepers. The parametric study provides design flexibility and choices to engineers. The outcome of
533 this paper will enhance the safety of the track component, and railway sleeper manufacturers could use
534 the model to assess their product designs.

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539 Engineering Network” (www.risen2rail.eu) [30]. Also, the first author wishes to thank the China
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